SOLUTIONS MANUAL

THERMAL ENVIRONMENTAL ENGINEERING

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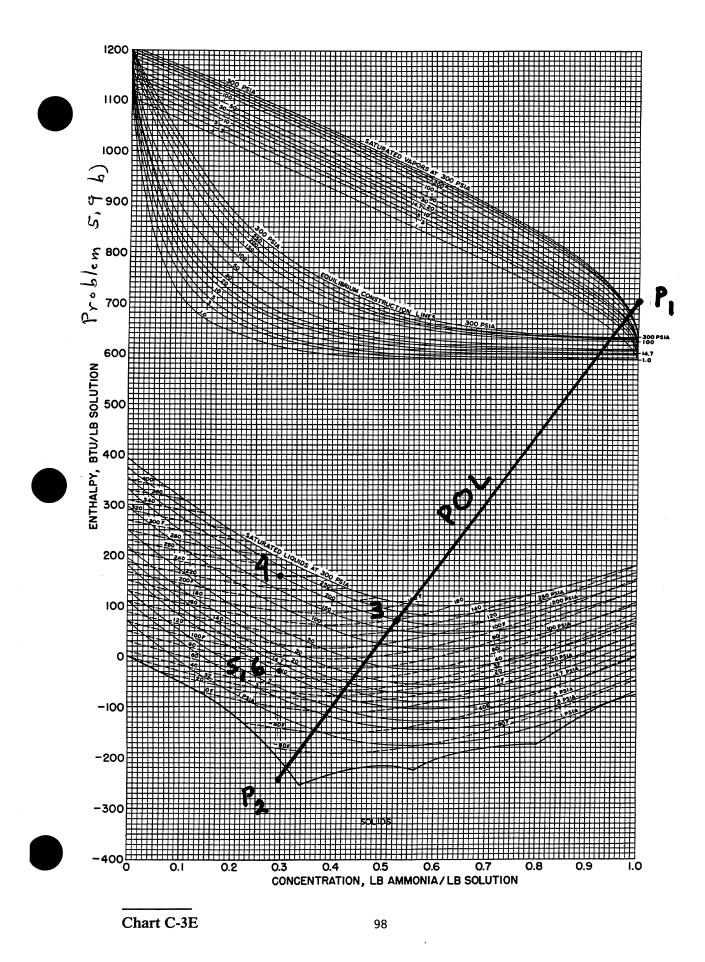
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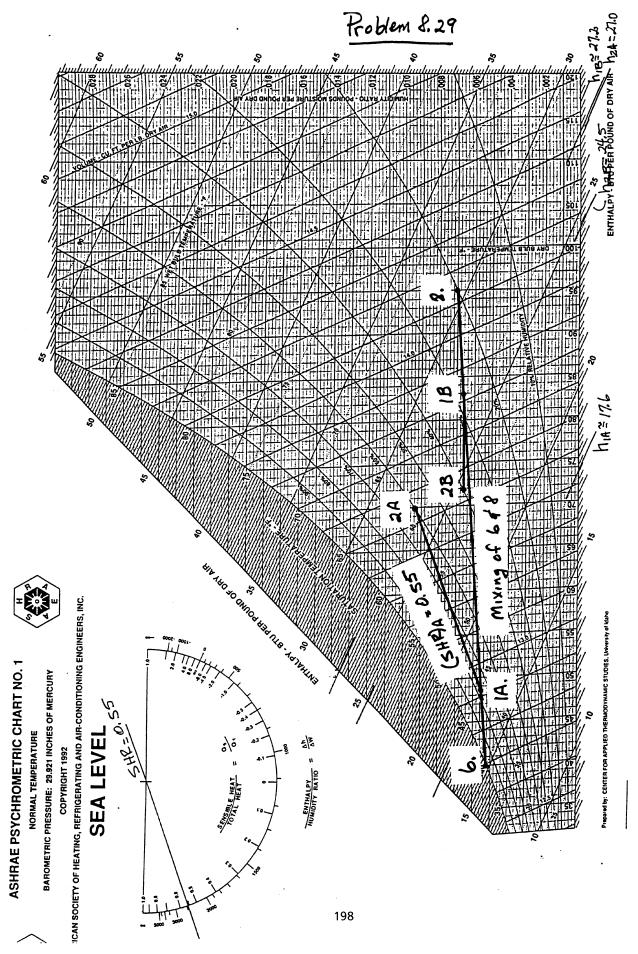
2.1
a)
$$28.75 \text{ in Hg} \left(\frac{1.450 \times 10^{-4} \text{ psia}}{2.961 \times 10^{-4} \text{ in Hg}} \right) = 14.08 \text{ psia}$$

b) 28.75 in Hg
$$\left(\frac{4.019 \times 10^{-3} \text{ in H}_20}{2.961 \times 10^{-4} \text{ in H}_9}\right) = 390.2 \text{ in H}_20$$

c)
$$\frac{390.23 \text{ in H}_20}{0.8} \left(\frac{1 \text{ m}}{39.37 \text{ in}} \right) = 12.39 \text{ m fluid}$$

d) 28.75 in Hg
$$\left(\frac{1 \text{ Pa}}{2.961 \times 10^{-9} \text{ in Hg}}\right) = 97.10 \text{ kPa}$$





5/5

11. 18

Summary of results

Component	Part(a)	Part(b)
Riw hr ft²/15ma	0.00923	0.00923
R _{Fi} w	0.01157	0.01174
Ro,w	0.03417	0.03 001
R _{t,w}	0.05497	0.05098
Ri, w / Rt, w	16.87.	18.17.
R _{F,w} / R _{t,w}	21.070	23.07.
Ro,w/Rt,w	62.2 %	58.973
Voiw Btu/hrft2 (Btu/Ibma)	18.2	19,6

14.20 The 97.5% design temp. for Chicago is -4°F (Table B.1)

a.) <u>Walls</u>	nal Resistance	hr-ff? of Btu
Source	At studs	Between Stude
Inside air film	0.68	0.68
3/8-inch gypsum	0.32	0.32
Studs	6.875	_
Insulation		19
25/32-inch sheathing	2.06	2.06
1.5-inch polystyrene*	7.5	7.5
4-inch brick **	0.6	0.6
Outside air film	0.17	0.17
Totals.	18.21	30.33

* Arbitrarily selected extruded polystyrene ** Arbitrarily selected lowest R-value/inch

Using the parallel path method, By Egns 14.18 and 14.12

$$(U_{av})_{wells} = \frac{0.25}{18.21} + \frac{0.75}{30.33} = 0.038$$
 Btu/hr.ft².oF

Windows: $U_0 = 0.33$ from Table 14.5

Roof: $U_R = 1/9.0 = 0.111$ Bta/hr.ft² of

Doors: $U_D = 0.20$ from Table 14.7

A windows = 700 ft^2 , $A_{doors} = 40.5 \text{ ft}^2$ A walls, frame = $(50)(10)(4) - 700 - 40.5 = 1259.5 \text{ ft}^2$ A Roof = 2500 ft^2

Problem 17.13 Bin Calculation.	Bin Calculatio		Denver Weather Data. Air to Air Heat Pump Heating	Air Heat Pump	Heating			
					•			
(UA)effective = Btu/hr	1122	This is the effe	1122 This is the effective (UA)-value for the building	or the building				
ti, °F=	89	68 Indoor temperature	ıture	,				•
internal gain= Btu/hr	0006	This is the sum	9000 This is the sum of internal gains expected	expected				
Heat Pump Capacity=	Qcond =	$00 + 385t_o + 2.54t_o^2$.54t _o Btu/hr					
DC	0.25 Thi	This is the part	is is the part load degredation coefficient	n coefficient				
Power W Requirement=	$W_{comp} = 1.28 +$		$0.0093t_o - 0.000085t_o^2$ k	kW			-	
tz, °F=	60.0 Thi	This is the calc	s is the calcuated Zero-load temp	emp.				
A	Δ.	ပ	٥	Ш	u.	g	I	
Avg. Bin Temp Annual Hours		Heating Systen Heat Pump	Heat Pump	Part Load facto	Run Time	Compressor Power	Backup Energy	Fnerov Input
J		Load, Btu/hr	Capacity, Btu/hr	_	Hrs	κw	kWh	KWh
		(can't be <0)		PLF eqn	B*C/(D*E)		(C-D)/B)/3412	(F*G)+H
				(can't be > 1)	(can't be > col. B)	3)	(can't be < 0)	
62	194	0			0	1.53	0	0
57	776						0	109
52							0	287
47							0	473
42							0	069
37	704	25/82	34922	0.935	575	1.51	0	1031
22							122	
22								
17	243			-			1691	2034
12		53832	22186	•	137	1.38	1271	1460
2	84	59442	20019			1.34	971	1083
2	54	65052	17980	-	54	1.30	745	
6-	22	70662	16068	-		1.25	352	
8-	13	76272	_	_	13	1.20	236	
-13	5	81882	•	1.000	5	1.14	101	107
-18	3	87492	11093	1.000	8	1.09	29	70
-23	1	93102	9689	1.000		1.02	24	
						Annual Ene	Annual Energy Input in kWh=	14108